

CANTILEVER

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A cantilever is a thin uniform bar fixed horizontally at one end and loaded at other end.

Two cases:

1. Bending of the beam due to the weight of the beam is not to be considered.
2. Bending due to the weight of the beam also be considered.



Case1:When the weight of the beam is ineffective

Let AB represents the neutral axis of a cantilever of length L, fixed at the end A and loaded at end B with a weight W. The end B is deflected into the position B' and the neutral axis takes the position AB'.

Consider a section PQ of the beam at a distance x from the fixed end A.

Deflecting couple due to the load W at this section= $W(L-x)$

Restoring couple= YI/R

At equilibrium $W(L-x)=YI/R$



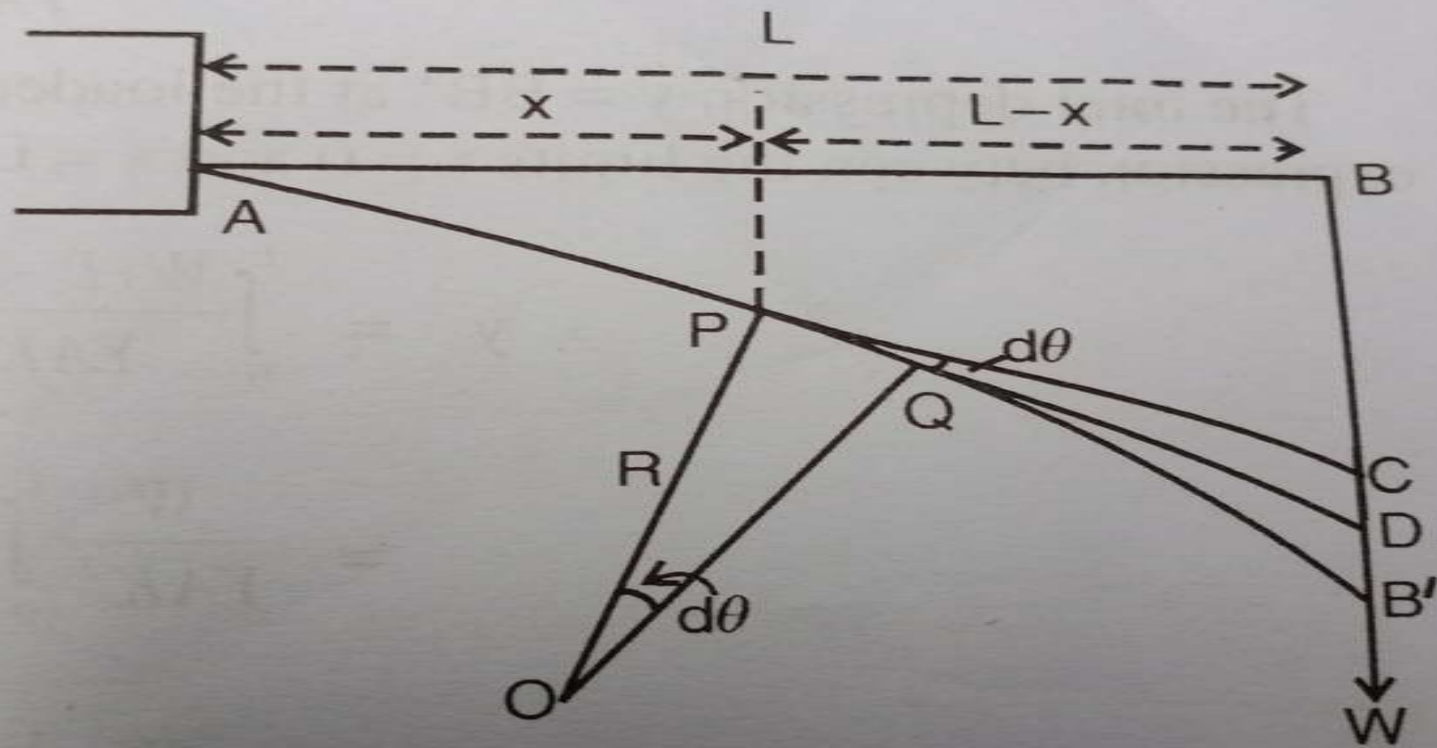


Fig.8

$$W(L-x) = YAK^2/R \text{ -----(1)}$$

From the figure , $dx = R d\theta$ or $R = dx/d\theta$

$$W(L-x) = YAK^2 \frac{d\theta}{dx}$$

$$d\theta = \frac{W(L-x)dx}{YAK^2} \text{ -----(2)}$$

The tangents from P and Q meet at C and D on the vertical line BB'. The angle subtended by the tangents is also equal to $d\theta$.

The depression CD can be represented by dy .

$$dy = (L-x)d\theta$$



Substituting from equation(2)

$$dy = \frac{(L-x)W(L-x)dx}{YAK^2} = \frac{W(L-x)^2 dx}{YAK^2}$$

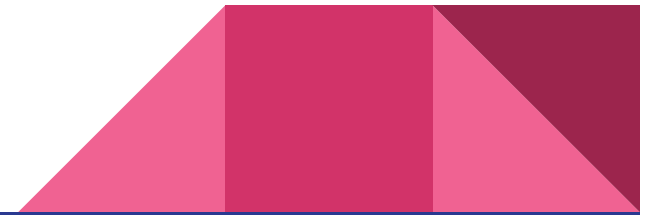
The Total depression $y=BB'$ at the loaded end B is obtained by integrating the above expression between the limits $x=0$ and $x=L$

$$\therefore y = \int_0^L \frac{W(L-x)^2 dx}{YAK^2} = \frac{W}{YAK^2} \int_0^L (L^2 - 2Lx + x^2) dx$$

$$y = \frac{W}{YAK^2} \left[L^2 x - \frac{2Lx^2}{2} + \frac{x^3}{3} \right]_0^L$$

$$y = \frac{W}{YAK^2} \left[L^3 - L^2 + \frac{L^3}{3} \right]$$

$$y = \frac{WL^3}{3YAK^2} = \frac{WL^3}{3YI}$$



When the weight of the beam is not considering ,

The free end of the cantilever is depressed by $\frac{WL^3}{3EI}$



Case2:Cantilever loaded uniformly

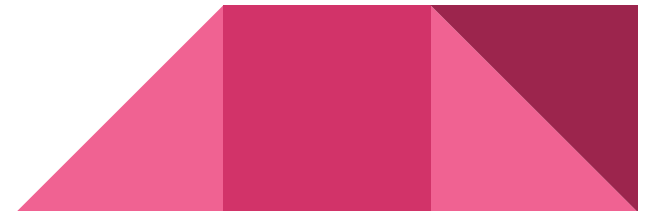
In this case, no weight is added to the end B. Let the weight per unit length is W_1 . The weight of the portion $(L-x)$ of the beam is $W_1(L-x)$.

The weight $W_1(L-x)$ acts at a distance $(L-x)/2$ from the section PQ. Bending moment due to weight of the beam is

$$W_1(L-x) \frac{(L-x)}{2} = \frac{W_1}{2} (L-x)^2$$

At equilibrium bending moment will be equal to the restoring moment

$$\frac{YI}{R} = \frac{W_1}{2} (L-x)^2$$



$$\frac{YI}{R} = \frac{W_1}{2}(L-x)^2$$

We know $R = \frac{dx}{d\theta}$

$$\frac{YI d\theta}{dx} = \frac{W_1}{2}(L-x)^2$$

$$d\theta = \frac{W_1(L-x)^2 dx}{2YI}$$

Also $dy = (L-x)d\theta$

$$\therefore \frac{dy}{(L-x)} = d\theta$$

Substituting $d\theta$, *we have*

$$dy = \frac{W_1(L-x)^3 dx}{2YI}$$



Total depression will be obtained by integrating the above expression within the limit $x=0$ and $x=L$

$$y = \frac{W_1}{2YI} \int_0^L (L-x)^3 dx = \frac{W_1 L^4}{8YI}$$

Since $w_1 L = W_2$, the weight of the beam

$$y = \frac{W_2 L^3}{8YI}$$



Special cases:

1. RECTANGULAR CROSS SECTION

Moment of inertia $I = \frac{bd^3}{12}$ where b=breadth and d=thickness of the beam

Depression at the loaded end,

$$y = \frac{WL^3}{3YI} = \frac{MgL^3}{3YI} = \frac{4MgL^3}{Ybd^3}$$



2. CIRCULAR CROSS SECTION

Moment of inertia $I = \frac{\pi r^4}{4}$ where r is the radius

Depression at the loaded end,

$$y = \frac{MgL^3}{3YI} = \frac{4MgL^3}{3Y\pi r^4}$$



ASSIGNMENT

Obtain the expression for y considering the weight of the beam and loaded weight at one end .



THANK YOU.....

